

# Product Instruction P92-9

## POSITIONER PT. NO. 5311450-□ APPLIED TO DIAPHRAGM-ACTUATED VALVES

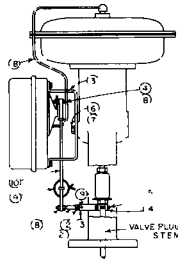
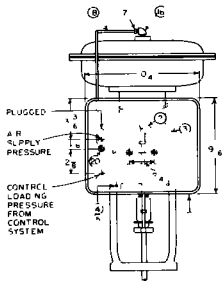
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Bailey Positioner Part Number 5311450-□ is a pneumatic relay device which is applied to a Control Valve to accurately position an inner valve in response to the control demand signal

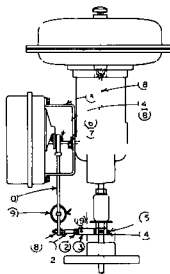
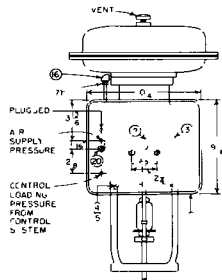
### CROSS REFERENCES

<u>Instrument or Equipment</u>	<u>Instruction Section</u>
Positioner Installation on Diaphragm-Actuated Valves	P92 32
Connecting Tubing	G18 2



TOP CONNECTED VALVE ACTUATOR  
TYPE V81120

TABLE A		
NAME	PART NO.	FOR
MOUNTING	53 450 2	5 25 PSIG OR 3 27PSIG
	53 450 5	3 5 PSIG
	53 726 A	3/4" A B
	53 739 B & C	1/2" STROKE
POSITIONING CAMS	53 449 A	1/2" 2 1/2" 3
	53 448 B	1/2" 2 1/2" 3
	53 54 C	1/2" 2 1/2" 3
POSITIONER DRIVE ARM	53 686	FOR ALL CAM STROKES SHOWN ABOVE
CLAMP	53 687 2	1/4" TO 3/4" O.D. VALVE STEM
CLAMP PLATE	53 243	1/4" O.D. VALVE STEM
	53 69	3/8" TO 3/4" O.D. VALVE STEM
CONNECTING LINK	53 247	3/4" TO 1" O.D. VALVE STEM 1/4" STROKE
	53 249 2	5/16" TO 3/4" O.D. VALVE STEM 1/4" STROKE
	53 249 7	5/16" TO 3/4" O.D. VALVE STEM 1/4" STROKE
	53 249 7	5/16" TO 3/4" O.D. VALVE STEM 1/4" STROKE



BOTTOM CONNECTED VALVE ACTUATOR  
TYPE V81020

TABLE A		
NAME	PART NO.	FOR
POSITIONER AND POSITIONING CAMS	SEE TABLE A	
POSITIONER DRIVE ARM	SEE TABLE A	
BRACKET	53 690	
NO. 0 3 1/2 LG. RD. HD. STL. SCREW		
NO. 2 1/4" NO. SHAKEPROOF LOCKWASHER		
5 6 8X 1/2 LG. HD. STL. SCREW		
NO. 2 8-00 SHAKEPROOF LOCKWASHER		
ELASTIC STOP NUT	97 20 5	
1/8" DIA. 6 T. PE. 316 STL. WIRE		
CONNECTING LINK	SEE TABLE A	
ADJ. DRIVE ARM	53 690	
1/4" 2 1/4" HEX. STL. NUT		
CLAMP	SEE TABLE A	
CLAMP PLATE	SEE TABLE A	
1/8" 8X 2 1/2 LG. HD. HD. COPPER SCREW		
ELBOW CONNECTOR	95 59 3	
CONNECTOR NUT	95 58 2	
1/4" O.D. COPPER TUBING		
1/4" X 3/8" LG. GROOV. P.N. TYPE		
MALE CONNECTOR	94 67 2	
NO. 2 2-00 SHAKEPROOF LOCKWASHER		

FIGURE 1 - Positoner Mounted on Diaphragm Type Actuator

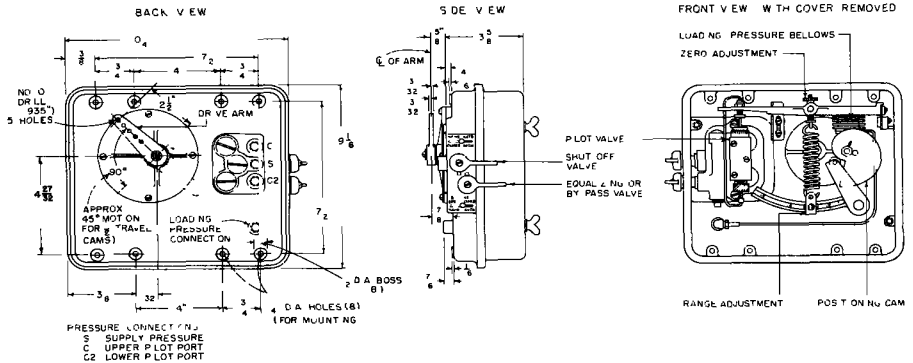


FIGURE 2 - Bailey Positioner

## INSTALLATION

If the Positioner is included with a Control Valve furnished by Bailey Meter Company, it is mounted on the valve yoke and piped to the diaphragm casing as shown in Figure 1. (Pressure gages are not included unless specified.)

If the Positioner is furnished separately for mounting on a valve, refer to Instruction Section P92-32 for installation procedures.

When making connections to the Positioner, refer to Instruction Section G18-2 for tubing

methods and precautions, and note the following (see Figure 2)

- 1 Positioner connection C1 applied to a Diaphragm Actuated Valve is always plugged.
- 2 Positioner connection C2 is always connected to Diaphragm Actuator casing.
- 3 Bypass valve and pilot valve block inner passages are arranged so that control pressure is always delivered thru connection C2.

## DESCRIPTION OF OPERATION

The Positioner (Figure 3) consists essentially of two opposing forces balanced against each other. When the Positioner is at balance, the force exerted upward on the balance beam by the loading bellows is equal to the downward force exerted by the positioning spring.

The force exerted by the bellows depends upon the control loading pressure established by the system in relation to the demand for the controlled medium. The force exerted by the positioning spring depends upon the position of the Valve stem (or position of the inner valve) and the shape of the positioning cam.

The Valve stem is tied back to the Positioner drive arm thru the Positioner drive rod. Therefore, for every position of the Valve stem, the

drive arm assumes a corresponding position. The drive arm is geared to the positioning cam which is shaped to give a desired characteristic of inner valve position vs control loading pressure.

### Automatic Positioning

Refer to Sketch A in Figure 4. When the control system indicates that the Control Valve should be opened, the force exerted on the balance beam by the loading bellows increases as a result of the increase in control loading pressure. Movement of the beam raises the pilot Valve stem, opening the upper valve port to full supply pressure, increasing the control air pressure applied to the diaphragm, and causing the actuator stem to move into the valve



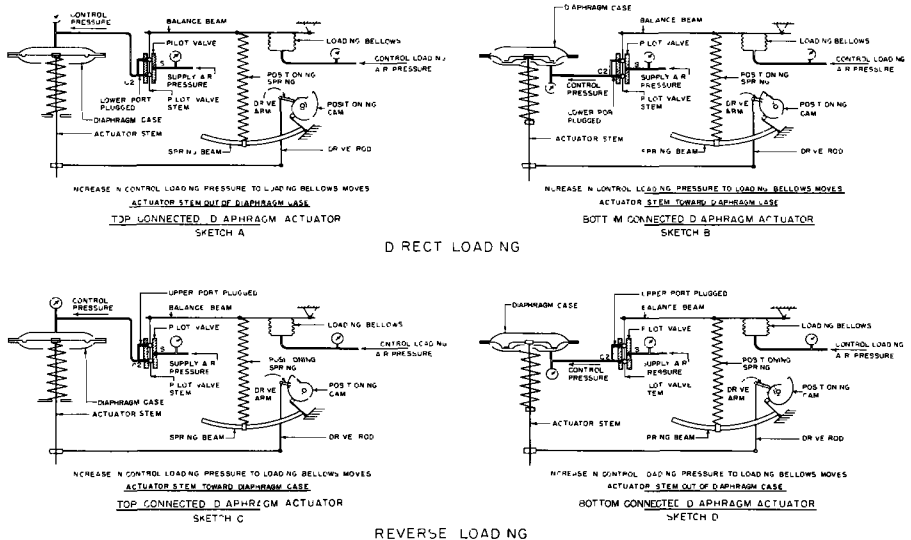


FIGURE 4 - Application of Positioner to Diaphragm Actuators

applied for direct loading, an increase in control loading pressure to the loading bellows causes an increase in control pressure to the Diaphragm Actuator. When applied for reverse loading, an increase in control loading pressure to the loading bellows causes a decrease in control pressure to the Diaphragm Actuator.

#### Direct Loading

1. **TOP CONNECTED DIAPHRAGM ACTUATOR** (Figure 4, Sketch A). An increase in control loading pressure to the loading bellows raises the pilot stem, increases control pressure to the Diaphragm Actuator, moves the actuator stem into the Valve body, lowers the Positioner drive arm, and turns the positioning cam clockwise to increase tension on the positioning spring until the forces exerted by the bellows and spring are at balance.

2. **BOTTOM CONNECTED DIAPHRAGM ACTUATOR** (Figure 4, Sketch B). An increase in control loading pressure to the loading bellows raises the pilot stem, increases control pressure to the Diaphragm Actuator, moves the actuator stem out of the Valve body, raises the

Positioner drive arm, and turns the positioning cam counterclockwise to increase the tension of the positioning spring until the forces exerted by the bellows and spring are at balance.

#### Reverse Loading

1. **TOP CONNECTED DIAPHRAGM ACTUATOR** (Figure 4, Sketch C). An increase in control loading pressure to the loading bellows raises the pilot stem, decreases control pressure to the Diaphragm Actuator, moves the Actuator stem out of the Valve body, raises the Positioner drive arm, and turns the positioning cam counterclockwise to increase the tension of the positioning spring until the forces exerted by the bellows and spring are at balance.

2. **BOTTOM CONNECTED DIAPHRAGM ACTUATOR** (Figure 4, Sketch D). An increase in control loading pressure to the loading bellows raises the pilot stem, decreases control pressure to the Diaphragm Actuator, moves the Actuator stem into the Valve body, lowers the Positioner drive arm, and turns the positioning spring until the forces exerted by the bellows and spring are at balance.

## OPERATION

After adjusting the Positioner for service as outlined on page 9, and with supply and control pressure applied, the Control Valve may be operated either manually or automatically as outlined below (see Figure 5)

### Direct Loading

#### To change from Remote Control to Local Manual Control:

- 1 Turn bypass valve to "Open Hand"
2. Turn supply valve to "Closed Auto".

#### To change from Local Manual Control to Remote Control:

1. Turn supply valve to "Auto-Open"
- 2 Turn bypass valve to "Closed Auto"

Manipulate valves in this sequence to avoid a momentary pressure loss to the Diaphragm Actuator

When the valves are set for manual operation, control loading pressure goes to the loading bellows and also thru the bypass valve to the Diaphragm Actuator (Figure 5, Sketch B) Thus, the Actuator is supplied with control loading pressure directly from the control system

The Control Valve may be positioned either by loading pressure from the control system or, preferably, by manual operation of the Selector Station (if used) connected by the control loading pressure line to the Positioner (see control or piping diagram of the specific control system of which the Control Valve is a part)

Note that the Positioner normally cannot be transferred from automatic to manual operation without disturbing the control system since the Positioner is usually calibrated to deliver control pressure to the Diaphragm Actuator which differs from the control loading pressure received from the control system (see "Calibration Adjustments", page 9) To maintain stable control while on manual operation, the Control Valve should be positioned by manual operation of the Selector Station to duplicate the characteristic provided by the Positioner on

automatic operation When the Control Valve is positioned by the handjack so that Positioner balance beam (Figure 6) is mid way between upper and lower stops, automatic Positioner operation is duplicated

#### To manually operate Control Valve by handjack:

- 1 Pick up Valve position with handjack
- 2 Turn supply valve to "Hand-Closed"
- 3 Turn bypass valve to "Closed Auto"

### Reverse Loading

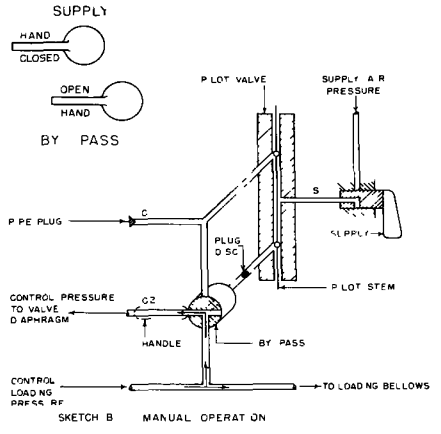
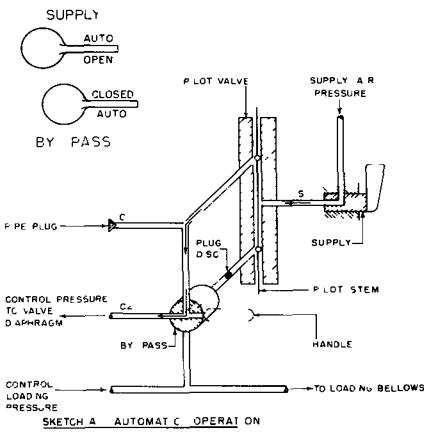
#### To change from Remote Control to Local Manual Control

- 1 Pick up Valve position with handjack.
- 2 Turn supply valve to "Hand-Closed"
- 3 Use handjack to position Control Valve

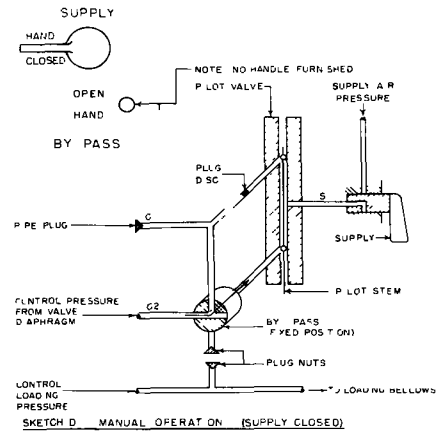
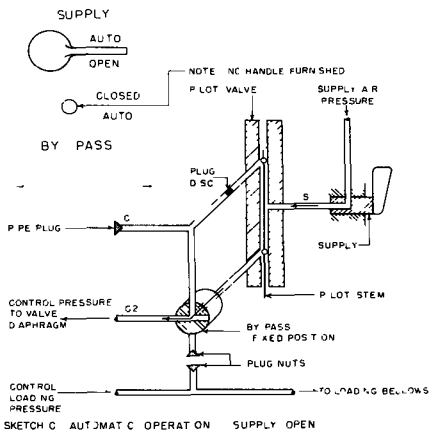
#### To change from Local Manual Control to Remote Control

- 1 Adjust remote control signal to agree with valve position in local manual control (Positioner balance beam will be midway between stops)
- 2 Turn supply valve to "Auto-Open"

NOTE: When arranged for reverse loading, the upper pilot valve port and the connection between the control loading pressure line and the Positioner bypass valve are plugged (see Figure 5) In addition, the bypass valve position is fixed and the valve handle is removed Serious trouble might result if the bypass valve position were altered and control loading pressure were introduced directly to the Diaphragm Actuator, since for reverse loading, the control pressure to the Diaphragm Actuator during automatic operation is the opposite of control loading pressure from the control system to the Positioner bellows, i.e. for an increase in control loading pressure, the control pressure to the Diaphragm decreases Thus, to effect a bypass arrangement, it is necessary to reverse the control loading pressure during manual operation of the Positioner This move is not practical for the small amount of time that the Positioner would be on HAND during normal operation



TOP OR BOTTOM CONNECTED DIAPHRAGM MOTORS DIRECT LOADING



TOP OR BOTTOM CONNECTED DIAPHRAGM MOTORS REVERSE LOADING

FIGURE 5 Supply and Bypass Valve Arrangements

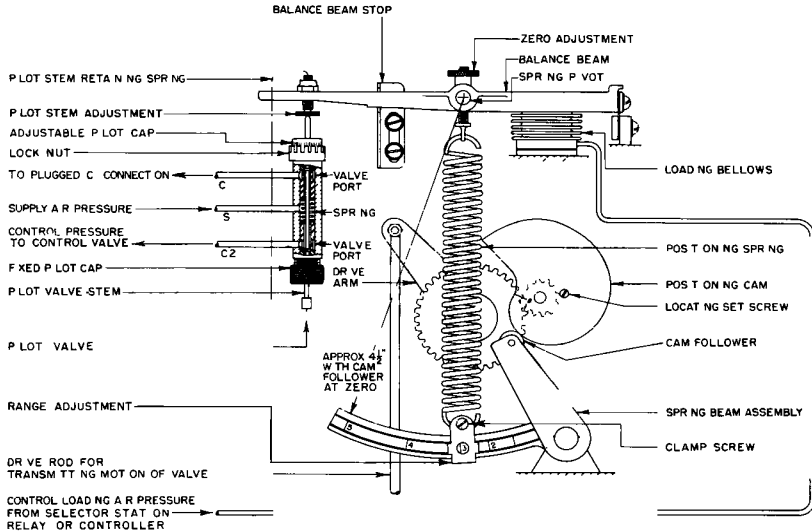


FIGURE 6 - Positioner Adjustments

## ADJUSTMENT AND CALIBRATION

Make the following adjustment checks to insure correct operation of the Control Valve and Positioner before attempting any adjustments to adapt the Valve to its particular application described under "Calibration Adjustments" page 9

### Adjustment Checks

The adjustments below are based on a direct loading, top connected Diaphragm Actuator and inner valve which opens as the Valve stem moves into the Valve body as shown in Figure 4. Sketch A

Normally, the Control Valve used in the direct loading arrangement will be in CLOSED position when the valve stem has traveled out of the Valve body to its fullest extent and OPEN when the stem has traveled into the Valve body to its fullest extent. Therefore, the words OPEN and CLOSED used below refer to the position in terms of each specific Control Valve. Refer to Figure 6

1 Use B or B 1/2 positioning cam which is shipped in place in the Positioner assembly (see "Characteristic Adjustment"), page 9

2 Make supply air (40 psig) connection at back of Positioner case (Figure 1)

3. Make certain that fixed pilot cap and adjustable pilot caps are tightened securely

4 Turn supply valve to "AUTO-OPEN" and bypass valve to "CLOSED AUTO". Set control loading pressure at zero psig.

5. If cam follower is not at zero mark on positioning cam with Valve in CLOSED position, adjust turnbuckle on drive rod until Positioner drive arm assumes the position which places follower on zero mark

6 Set loading pressure at minimum range value (3 psig). Control Valve should remain in CLOSED position and Positioner balance beam should be horizontal and between upper and lower stops. If beam is not horizontal, turn zero adjustment nut clockwise (in half turn increments) to lower beam to horizontal position or counterclockwise to raise beam to horizontal position.

7. Turn on air supply. Adjust pilot stem adjustment so valve is just closed.

8. Increase loading pressure to 3.5 psig. If Valve does not begin to leave CLOSED position as soon as pressure is increased, turn pilot stem adjustment nut until such movement is obtained.

9. Set loading pressure at maximum range value (15 or 27 psig). If Control Valve does not move to OPEN position, loosen range adjustment clamp screw and slide range adjustment along spring beam until Valve reaches OPEN position. Tighten clamp screw to insure that adjustment will remain locked.

10. Decrease loading pressure 0.5 psig below maximum range value (15 or 27 psig). If Valve does not begin to move from OPEN position as soon as pressure is decreased, re-adjust range adjustment until such movement is obtained.

11. Return control loading pressure to 3 psig. Valve should just go to CLOSED position and cam follower should be at zero mark. If not, repeat steps 6 thru 11 until desired valve travel is obtained.

### Calibration Adjustments

The Positioner adjustments described below may be used to improve the operation of the Control Valve either by itself or in relation to other systems or parts of a multiple system.

1. Zero or Suppression Adjustment - By means of the zero adjustment (Figure 6), an initial tension may be imposed upon the positioning spring so that the Valve will not start to move from its minimum position until the control loading pressure has increased from minimum range value (3 psig) to any value up to 40% of full loading pressure range (i.e., 3 to 15 psig for a 3-27 range or 3 to 8 psig for 3-15 range) or has decreased from maximum range value (15 or 27 psig) to any value down to 40% of full range, i.e., 27 to 13 psig (3-27 range) or 15 to 8 psig (3-15 range). This adjustment is of value when two or more Valves are to be operated in sequence or where the Valve is equipped with a minimum stop and its flow characteristic must be matched with that of another power device

NOTE: Suppression values above pertain to zero adjustment settings only when using full control loading pressure range and a full rise (uncut)

positioning cam. Greater suppression may be obtained from the "Range Adjustment" and "Characteristic Adjustment" below.

2. Range Adjustment - The range adjustment (Figure 6) affords a variation of Valve motion for a given range of control loading pressure. The variation extends, roughly, from full Valve travel for a 3 to 15 psig change in control loading pressure to one-half Valve travel for a 3 to 27 psig control loading pressure change. In combination with the zero adjustment described above, full Valve travel may be obtained for as small a loading pressure change as 10 psig (e.g., 10 to 20 psig range). Range adjustments available with each Positioner cam are shown in Figures 7, 8, and 9. This adjustment is of value when the Valve is oversized since it allows operation of the Valve thru its useful motion for the desired full change in control loading pressure. It is also useful in matching the loading vs position characteristic of the Valve with those of related power devices in the same control system.

3. Speed Adjustment - When the system involves a single Valve, a high Valve positioning speed is usually an advantage. In a complex control system it is generally desirable to operate all power devices at the same speed in order to avoid interaction between units or undesirable process conditions during load changes. The speed of operation of the Valve may be decreased as described on page 12 under "Speed Control".

4. Characteristic Adjustment - This adjustment involves selecting or shaping the proper positioning cam in order to obtain that characteristic of inner valve position to control loading pressure which will afford the desired characteristic of flow (or pressure drop, etc.) vs control loading pressure.

Three cams are furnished with each Positioner. Cams A, B, and C are furnished for Control Valves which have a stroke of 1-1/2 inches, 2-inches, or 2-1/2 inches and cams A-1/2, B-1/2, or C-1/2 are furnished for Valves with 3/4-inch stroke. The characteristics for which the cams are shaped are listed in Table 1 and are shown in Figures 7, 8, and 9. The Figures show a family of curves for each cam, each curve of the family representing a range adjustment when used with that specific cam. Table 2 shows pressure values of the various control loading ranges equivalent to the control loading per cent values shown in Figures 7, 8, and 9.

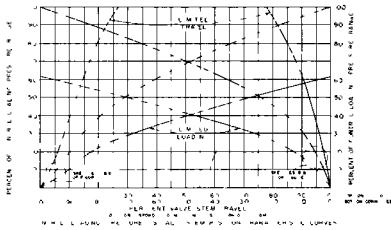


FIGURE 7 Cam A or A 1/2, Square Root Relation

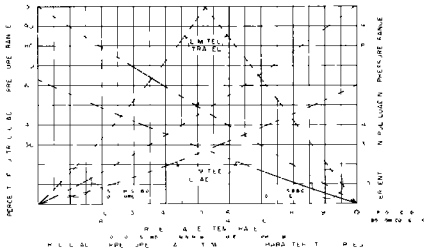


FIGURE 8 - Cam B or B-1/2, Straight Line Relation

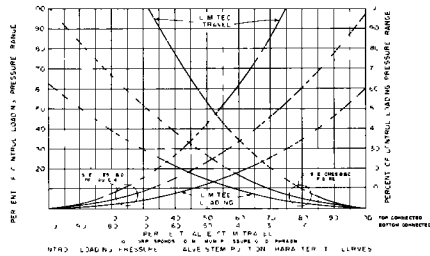


FIGURE 9 Cam C or C 1/2, Square Relation

Positioning Cam, Any Stroke	Inner Valve Position (P) vs. Control Loading (L)	Fig No
A or A 1/2	Square Root ( $L = \sqrt{P}$ )	7
B or B-1/2	Straight Line ( $L = P$ )	8
C or C-1 2	Square ( $L = P^2$ )	9

TABLE 1 Positioning Cam Characteristic

When a control system involves a single Valve, the B or B 1/2 straight line cam will probably be satisfactory. However, one of the other cams may provide a more uniform flow vs control loading characteristic and thus provide stable control over a wide range of operation. For a Valve which is an integral part of a complex control system, the cams provide a selection of characteristics which, together with the range adjustment, afford close paralleling of the position vs loading characteristics of the Valve with that of other power devices in the system. Refer to "Characterized Cams", below for a Valve which is to be part of a complex control system

CONTROL LOADING PRESSURE		
PER CENT VALUE	PRESSURE VALUE (PSIG)	
	Control System Ranges	
	3 27	3 15
0	3 0	3 0
10	5 4	4 2
20	7 8	5 4
30	10 2	6 6
40	12 6	7 8
50	15 0	9 0
60	17 4	10 2
70	19 8	11 4
80	22 2	12 6
90	24 6	13 8
100	27 0	15 0

TABLE 2 Conversion Table for Control Sys'em Ranges

Characterized Cams

In order to match the inherent characteristic of a specific inner valve to that of another inner valve of a different type, a damper, or a variable speed control etc., it is usually most practical to reduce the flow vs valve position characteristic of each device in the system to a straight line relationship with regard to control loading pressure. This straight line relationship is established by calibrating the Positioner with respect to the correct positioning cam by the following method

NOTE: In this procedure (and "Cam Shaping Method" page 11) substitute control air pressure for valve stem travel, substitute per cent rating or load (for boilers) for per cent flow. Note this substitution in the graphs of Figures 7 thru 13

1 Use the straight line cam B or B-1/2 to determine the actual flow vs valve position characteristic (see Figure 10). (The per cent of control loading range with the B or B-1/2 cam is equivalent to the per cent of Valve position )

2. Decide upon the exact flow vs control loading pressure characteristic desired (see Figure 11).

3 Take values for per cent valve stem travel from step 1 (Figure 10) and for per cent control loading pressure from step 2 (Figure 11) and plot a curve as shown in Figure 12.

4 Compare the curve made in step 3 with those shown in Figures 7, 8, and 9 and select positioning cam whose characteristic most closely matches the characteristic plotted in step 3

5 If necessary, set range and zero adjustments to match loading vs Valve position characteristic more accurately.

6. If the required characteristics cannot be matched by the above procedure or if a more exact characteristic is required, alter cam shape as described under "Cam Shaping Method"

#### Cam Shaping Method

To assist in the alteration process, cams are marked with radial lines (index of per cent travel) and concentric lines (index of per cent loading pressure) The ten concentric lines on the cam correspond to actual control loading pressure shown in Table 2 for the specific control system range being used.

NOTE: Before cutting any cam, make certain that cutting will involve removal of cam material rather than building it up. Thus, if the characteristic plotted lies between the A and B cam (Figures 7 and 8), the A cam should be cut, etc Alter cam shape as follows:

1 On cam selected in step 4 under "Characterized Cams", for each increment of control loading pressure (concentric lines), locate that Valve position (radial lines) which corresponds to flow required for specific loading pressure determined in step 3 above. Refer to Figure 13 for method of locating these points on cam

2 A curve drawn thru points located on cam in step 1 is desired cam shape Either alter cam or cut a new cam to this shape

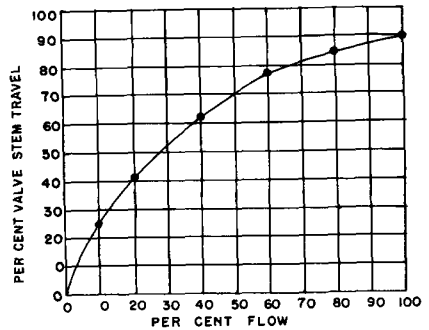


FIGURE 10 - Valve Characteristic

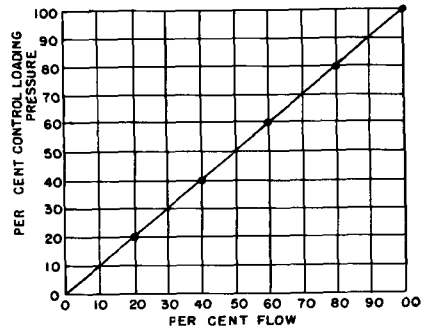


FIGURE 11 Desired Control

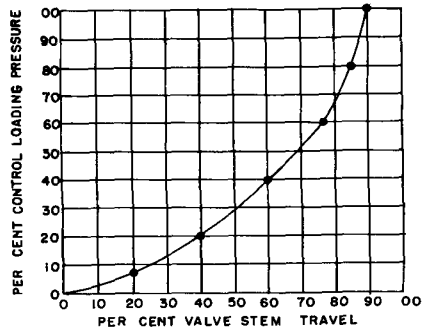


FIGURE 12 Cam Characteristic

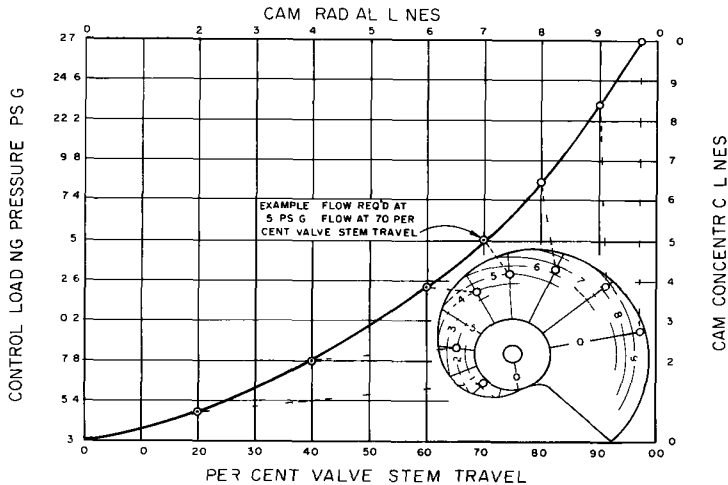


FIGURE 13 Locating Points for Shaping Positioner Cam

### Speed Control

If it is necessary to reduce the speed of Control Valve operation, a speed control orifice is available for insertion in the control line from the pilot valve to the Diaphragm Actuator. Note the time required for full travel (3/4, 1-1/2, or 2 inch valve stroke) If this time were 10 seconds, the orifice (No 60 drill, 040 inch diameter) would reduce the speed of operation to 24 seconds. The speed may be increased up to the normal in proportion to the diameter of the orifice hole. To insert a speed control orifice, follow the steps below

1 Remove pilot stem by springing open stem retaining spring and allowing stem to drop out (Do not scratch stem lands during removal.)

2 Remove pilot valve by removing attaching screws (see Figure 14).

3 Remove O ring gasket from upper or lower hole in pilot valve block that does not contain a plug disc

4. Insert speed control orifice in valve block hole from which O ring gasket was removed.

5 Replace O-ring gasket

6. Reassemble pilot valve to valve block

7 Reassemble pilot stem in pilot valve and replace stem retaining spring

## MAINTENANCE

### General

1. Keep air connections tight to prevent leakage which may indicate improper functioning of unit. Check all connections for leakage while under pressure with a soapsuds solution.

2. Maintain a clean air supply (free of dirt, oil, or moisture) for satisfactory operation of Positioner and Diaphragm Actuator.

### Routine Maintenance

1. Whenever Control Valve is out of service (or when required) remove pilot valve stem and inner liners and clean with a common solvent. Never use files, reamers, or abrasives on valve stem lands or valve liners. If liners stick in valve body upon removal, push them free with a wooden stick or pencil. never use a metal rod for this purpose

2. Once each year or whenever Positioner supply or bypass valves begin to show signs of sticking, remove valve and lubricate with Bailey Petcock Lubricant (Part No. 19987-1). To remove valve, leave handle in present position

and turn valve nut (Figure 12) out of pilot valve body.

NOTE: Valves are not interchangeable with each other or with valves in other Positioners. Valve and valve bodies are marked "1" and "2" for identification in reassembly.

3. Apply a few drops of light machine oil to spring beam assembly bearing at three or four-month intervals.

4. Once each year check filters in pilot valve body (Figure 14) and replace if dirty (note that filters are included as added protection and are not a substitute for the required clean air supply).

5. Once every two or three years change grease in Positioner gear case. Fill gear case about half full with aluminum stearate base grease with a 1-2 consistency (such as Vulcan Lube, Grade No. 1, from C. H. Clark Oil Co. Cleveland, Ohio). Rotate gears to work grease into teeth.

6. Once each year check calibration of Positioner

## REPLACEMENT PARTS

### Spare Parts Kit

The Spare Parts Kit shown in Figure 14 should be carried in stock. Specify the Spare Parts Kit part number to order a complete kit

### Ordering Individual Parts

Figure 14 is a Parts Drawing of the Positioner, Pt No 5311450-□. Normally this Drawing will apply to the unit furnished. However, there may be individual differences in specific units because of

a. Design changes made since the printing of this Instruction Section

b. Special design of the Positioner to make it suitable for a special application

Therefore, when ordering parts assure the receipt of correct replacements by specifying on the order

1. The complete nomenclature (stamped on instrument nameplate) of the Positioner for which parts are desired.

2. The Parts Drawing on which each part is illustrated. (The Parts Drawing Number is given in the title for the Figure )

ITEM	PART NO	NAME	ITEM	PART NO	NAME	ITEM	PART NO	NAME
1	CCIE LABEL	GIVE NO LABEL WHEN ORDERING PFS	30	5311448 1	ADJUSTING NUT	65	SEE TABLE	PILOT VALVE ASSY INCL
2	90171 4	TEE SEE TABLE	31	5311439 1	EYE BOLT	531028	5311038 1	CAP FOR 1 8 PILOT VALVE
3	5311031 1	SPRING RUPPORT	32	5311442 1	ADJUSTING PIVOT SPRING	531077 2	5311038 1	CAP FOR 5 16 PILOT VALVE
4	53 2092 1	SLEEVE	33	SEE TABLE	SPRING	531077 2	5311038 1	VALVE SEAT 2 REQ FOR 1 8 PILOT V
5	8 32 1 2	FIL HD STN STL SCR	34	5311422 1	BELLGOWN & TUBE ASSY	65C	5311038 1	SLEEVE 2 REQ FOR 1 8 PILOT V ONLY
6	5311254	TUB ASSY SEE TABLE	37	5311429 1	CAM SHAFT & PINION	65D	5311426 1	P LOT VALVE SPG
7	U 32x3 8	EXT STN STL SEMB 11 REQD	38	531 415 1	SPRING RETAINER	65E	5311408 1	VALVE BODY
8	197227 1	SPECIAL HEX HD SCR	39	5311414 1	P LOT BEAM SPRING	65F	536448	CAP LOCK NUT
10	5311686	POS TIONER DR ARM	40	5311675 1	SPRING RETA NER	65G	536467 1	ADJ CAP
11	5311616 1	SPEED CONTROL WASH	41	5 16 18 10	HEX JAM STN STL NUT	66	10 328 8	EXT HD HDFTN STL SEMB 3 REQD
12	531 147 1	PILOT VALVE GASKET	42	197165 1	SET SCREW	67	53 1419 1 8	F LTER CAP 3 REQ
13	J 1304 1 8	COVER EASY INCL	43	97 64 37	RETA NING RING	68	5311426 1	O RING GASKET 2 REQ
14	5311407 1	SEAL N 5 8 NIP	44	53 1634 1	CAM RUD	69	53 402 1	CONN PLATE SEE TABLE
14A	5311408	COVER	45	SEE TABLE	SQUARE CAM C	70	53 1476 2	FIL WASH & D N ASSY 3 REQ
48	5311409 1	COVER SCR 3 REQ	46	SEE TABLE	SQUARE SECT CAM A	71	5313000 1	F LTER CUP 3 REQ
14C	149173 10	RET RING 2 REQ	47A	531214 1	REAR REG COVER NCL	73	5311427 1	FILTER SPY 3 REQ
15	8 32x5 16	RD HD STN STL SCREW 7 REQD	47B	531121 2	NEEDLE BEARING	74	1 76 27 NPT	S 2 HD STN BRASS PIPE PLUG
16	NO 5	PLA N STN STL WASH 2 REQD	5	SEE TABLE	NAMEPLATE	75	3 4867	BAILEY PETCOCK LUBRICANT
17	531103 1	HANDLE 2 REQ	2	NC 0x1 8	RD HD STN STL THD FORMING SCR TYPE 4 REQD	76	SEE TABLE	BACKPLATE ASSY
20	5311418 1	PILOT BEAM ASSY	53	NO 1 4 20	EXT NEPS LA WASH STN STL			
21	531 456	P LOT STEM SPRING	54	10 32x7 8	EXT RD HD SEMB STN STL			
22	4 20x3 8	RD HD STN STL SCREW 7 REQD	55	53 4543 1	SP CAM MTG BRKT			
	53 1451 1	PILOT STEM ADV SCR	56	0 37	HEX STN STL NUT			
24	53120 2	SEAL 7 8 TUBE	58	53 079 1	TURBISON SPRING			
25	53 1433 1	GEAR & SHAFT ASSY	59	531264 1	SPRING			
26	SEE TABLE	PILOT STEM	60	531761 2	LATCH			
27	53 1423 1	SPG BEAM ASSY DN L	61	4804 09 0 4107	FLTR WASH STN STL 2 REQD			
27A	5311424 1	CORING RING 2 REQ	62	185 87 1	EYE BOLT			
27B	531 426	CAM ROLLER	63	SEE TABLE	PILOT V & B/LK ASSY INCL			
27C	197164 15	RET RING 2 REQ	64	SEE TABLE	P LOT V B/LK ASSY INCL			
27D	5311446 1	CAM ROLLER PIN	64A	5311301 1	VALVE NUT 2 REQ			
28	53114 7	P LOT BEAM STOP	64B	53 401 3	SPRING 2 REQ			
29	NO 1610 00	SHK LA WASH STN STL						

TABLE OF VARIABLE PARTS

P art No	U-GAL	P LOT	CAM	RANGE	ITEMS	ITEM 26	ITEM 26*	ITEM 33	ITEM 45	ITEM 48	ITEM 51	ITEM 63	ITEM 64	ITEM 65	
53 454 1	DRIVE	1 8	FULL	3 27 5 25	OMIT	53 4 2 1	5311439	53 444 1	5311446 1	531 341	5311448 1	18670 1	53 1397 1	5318050	53149 9 1
531450 2	VALVE	1 8	HALF	3 27 5 25	1 REQ	53 4 2	5311439 1	53 444 1	5311799 1	531 798 1	5312285 1	18660 1	53 1397 3	5310350 2	53148 9 1
53 1430 3	DRIVE	3 4	FULL	3 27 5 25	OM T	5311410 1	53 827	5311444 1	531 448	531344 1	5311449	18660 1	531109 2	53180 1	53148 9 1
53 145 4	DRIVE	1 8	FULL	3 5	OM T	5311412 1	53 439 1	53 670 1	531 448	531344 1	5311449 1	18660 1	531 39	53160 1	53148 9 1
53 450 7	VALVE	1 8	HALF	1 5	1 REQ	531 412	5311439	53 878	531 799	53 98 1	5312276 1	18660 1	531136 2	53 8050 2	53148 9 1
531 450 6	DRIVE	3 4	FULL	3 5	OM T	5311412	53 877 1	531678 1	50 448	531 341 1	53 449	18660 1	53 397 2	53170 1	53148 9 1
5311451	DRIVE	1 8	FULL	1 5	OMIT	53 41 1	531474	531876	5311448 1	53 244	531 341	18660 1	531 39	53105 1	53148 9 1
53 450 8	DRIVE	1 8	FULL	3 5	1 REQ	5311412	53 1871 1	5311444 1	5320074 1	5311449 1	18660 1	5311367 1	53 8050 3	53 4880	53148 9 1
531 450 4	VALVE 3 16	HALF	3 27 5 25	1 REQ	53 4 1	5311412	53 877 1	531882 1	53 679 1	531 798 1	53 789	53 789	531176 2	531090 1	53148 9 1
531450 11	VALVE 1 8	HALF	3 27 5 25	1 REQ	53 4 1	5311412	53 1459 1	5311444 1	5311444 1	53 789	53 789	53 276	18660 1	53 360 1	53148 9 1
53 450 17	VALVE 1 8	FULL	3 27 5 25	1 REQ	53 14 2 1	5311439	53 444 1	5311449 1	53 244	531 341 1	5311449 1	18660 1	53 1397 2	53160 1	53148 9 1
531 400 12	VALVE 1 8	FULL	3 5	1 REQ	53 14 2 1	5311439	53 444 1	5311449 1	53 244	531 341 1	5311449 1	18660 1	53 1397 1	53160 1	53148 9 1
531 470 14	DRIVE 1 8	FULL	3 5	1 REQ	53 14 2 1	5311439	53 444 1	5311449 1	53 244	531 341 1	5311449 1	18660 1	53 1397 1	53160 1	53148 9 1
5311450 5	DRIVE 1 8	FULL	3 27 5 25	OM T	532108	5314 7	53 144 1	53 448 1	5311449 1	5311449 1	18660 1	531139 1	53 8050 1	53148 9 1	

SPARE PARTS KITS

KIT NO 2 7 0					KIT NO 56001 2					KIT NO 7400 3				
FOR POS TIONER PART NOS 5311450 1 4 5 11 12 J					FOR POS TIONER PART NOS 531 450 3 6 0 0					FOR POS TIONER PART NOS 53 450 7 8 4 15				
( 8 PILOT S Z E					3 1 8 PILOT S Z E					1 8 PILOT S Z E				
QUANTITY	ITEM NO	SEE NOTE	ITEM NO	SEE NOTE	QUANTITY	ITEM NO	SEE NOTE	QUANTITY	ITEM NO	SEE NOTE	QUANTITY	ITEM NO	SEE NOTE	
EACH	20 26 2 B 77D 38 44 74 53 65E 75				EACH	20 26 2 B 77E 38 44 74 53 65E 75		EACH	20 26 2 B 77B 29 34 44 74 53 65E 75		EACH	20 26 2 B 77B 29 34 44 74 53 65E 75		
2 EACH	77C 65B 74				2 EACH	2 L 65B		2 EACH	2 L 65B 65L		2 EACH	2 L 65B 65L		
2	33 SEE NOTE 7				2	33 SEE NOTE 7		2	33 SEE NOTE 7		2	33 SEE NOTE 7		
3 EACH	68 70 71 73				3 EACH	68 71 71 73		3 EACH	68 70 71 73		3 EACH	68 70 71 73		

NOTES  
 WHERE THE ITEM SPEC F ED 5 A VAR ABLE PART THE PART APPLICABLE TO THE POS TIONER L STET IS CLUDED 7 FT  
 2 KIT INCLUDED RANGE SPRING 531 444 1 FOR S Z E 2 5 3 4 2 P S C AND 531616 1 FOR S Z E 5 6 5 G  
 \*FOR PISTO OPERATED VALVES SUBSTITUTE ITEM 531474 1 FOR ITEM 531439 IF HU T NO CL T OCCURS  
 \*\*DUAL OPERATR  
 \*\*\*SEE PILOT S Z E COLUMN IN TABLE FOR APPLICABLE P LOT VALVE S Z E

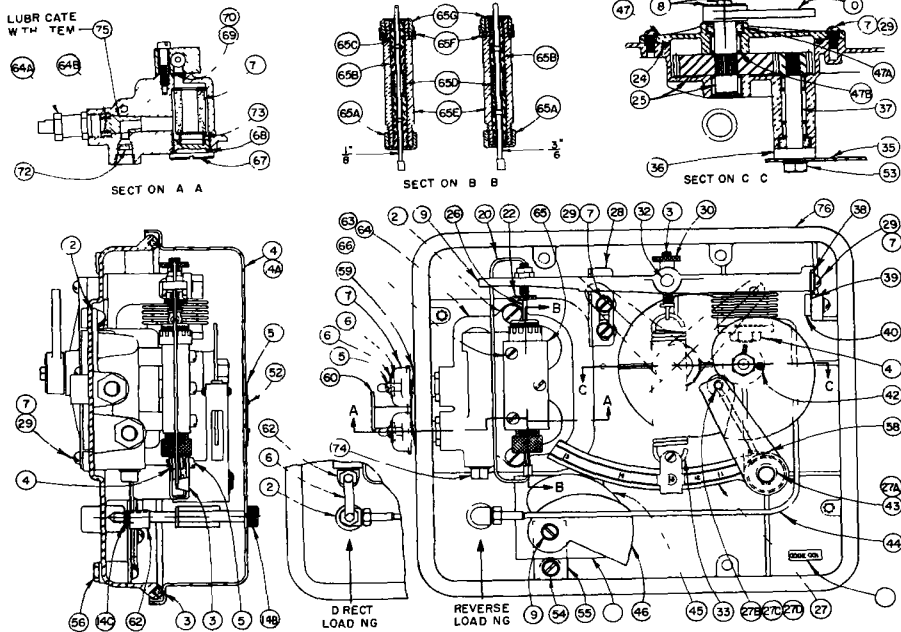


FIGURE 14 - Parts Drawing P92-11, Positioner Assembly

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